Q.P. Code: 19ME0318

R19

Reg. No:

SIDDHARTH INSTITUTE OF ENGINEERING & TECHNOLOGY:: PUTTUR

(AUTONOMOUS)

B.Tech III Year II Semester Regular Examinations August-2022 DESIGN OF MACHINE ELEMENTS - II

(Mechanical Engineering)

Time: 3 hours

Max. Marks: 60

L5

(Answer all Five Units $5 \times 12 = 60$ Marks)

UNIT-I

1 a Differentiate the straight and curved beams.

L2 2M

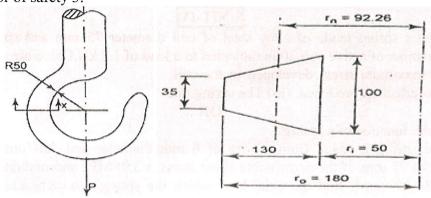
10M

12M

12M

L5

b A crane hook has a section, which for the purpose of analysis is considered trapezoidal as shown in fig. it is made of plain carbon steel with a yield strength of 350Mpa in tension. Determine the load capacity of the hook for a factor of safety 3.



- Two shafts whose centres are 1 metre apart are connected by a V-belt drive. The driving pulley is supplied with 95 Kw power and has an effective diameter of 300 mm. It runs at 1000 r.p.m. while the driven pulley runs at 375 r.p.m. The angle of groove on the pulleys is 40°. Permissible tension in 400 mm² cross-sectional area belt is 2.1 MPa. The material of the belt has density of 1100 kg / m³. The driven pulley is overhung, the distance of the centre from the nearest bearing being 200 mm. The coefficient of friction between belt and pulley rim is 0.28. Estimate:
 - i) The number of belts required; ii) Diameter of driven pulley shaft, if permissible shear stress is 42 MPa.

UNIT-II

3 Design a journal bearing for centrifugal pump from followingdata:
Load on the journal = 20 kN Speed of the journal = 900 rpm
Type of oil SAE 10 for which absolute viscosity at 55°C = 17centipoises
Ambient temperature of oil = 15.5°C
Maximum bearing pressure for the pump = 1.5 N/mm2
Calculate also the mass of the lubricating oil required for artificial cooling to rise in temperature of the oil limited to 10°C. Heat dissipation coefficient = 12.2 kN/m²/°C

OR

The ball bearing for the drilling machine spindle is rotating at 3000rpm. It is **L6** 12M subjected to radial load of 2500N and an axial load of 1500N. It is to work 50 hours per week for one year. Design a suitable bearing if the diameter of the spindle is 40mm.

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UNIT-III

The following data is given for the piston of a four-stroke diesel engine: Cylinder 12M bore = 250 mmMaterial of piston rings = Gray cast iron Allowable tensile stress=100N/mm² Allowable radial pressure on cylinder wall = 0.03 MPaThickness of piston head = 42 mm and No of piston rings = 4 mmCalculate: (i) Radial with of piston rings. (ii) Axial thickness of piston rings. (iii) Gap between the ends of piston rings before and after assembly. (iv) Width of the top land. (v) Width of the ring grooves. (vi) Thickness of the piston barrel and thickness of the barrel open end. L₂ a Explain why torsional vibrations are dangerous. **6M b** Explain reasons for the failure of a crank shaft. L₂ **6M UNIT-IV** A compression spring made of alloy steel of coil diameter 75 mm and spring 12M index 6.0, number of active coil 20 is subjected to a load of 1.2 kN.Calculate: (i) The maximum stress developed in the coil. (ii) The deflection produced. (iii) The spring rate. a What is the function of a spring? L13M**b** A helical spring is made from a wire of 6 mm diameter and has outside L5**6M** diameter of 75 mm. If the permissible shear stress is 350 MPa and modulus of rigidity 84 kN/mm², find the axial load which the spring can carry and the deflection per active turn. **UNIT-V** A compressor running at 300 rpm is driven by 15kW, 1200rpm motor 12M through 20⁰ full depth involute gears. The centre distance is 375mm. Choose the suitable materials for pinion and gear, design drive. 10 A pair of gears is to be designed to transmit 30kW for a pinion speed of 1000 12M rpm and a speed ratio of 5. Design the gear train.

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